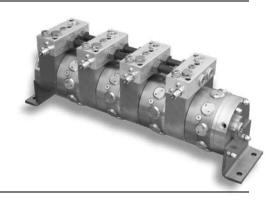
FLOW DIVIDERS FD/FDM SERIES TECHNICAL CATALOGUE



FD FDM



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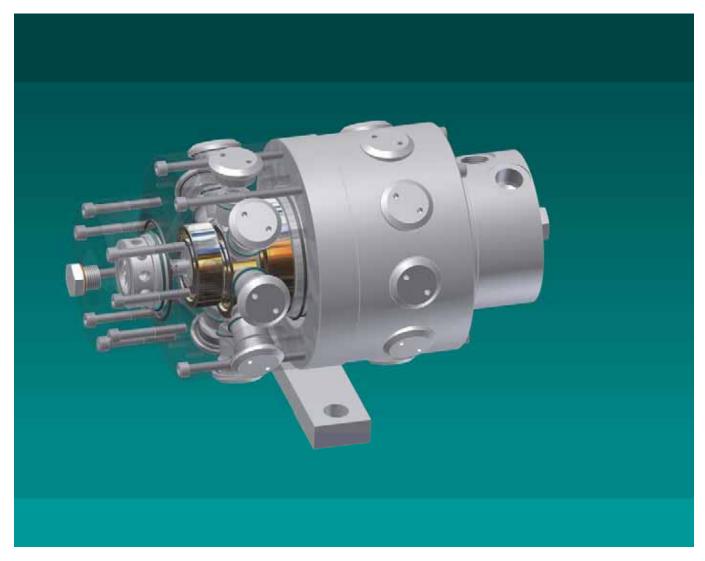
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GENERAL INFORMATION

INTERMOT has been manufacturing RADIAL PISTON HYDRAULIC MOTORS since 1985: our yearly production is more than 13.000 units which we sell all over the world through our agents and authorized sellers. Our motor range varies from 20cc to 8500cc displacement and it is completed by two-speed motors and special motors created in cooperation with our clients for different applications such as: underwater, high & low speed and wheel motors and with the possibility to assemble valves, brakes or gear reductions. You can directly contact our Technical Department which will give you all the necessary support to find the right solutions to your problems.

INTERMOT is a flexible work reality and manages deliveries also within the same day of order; we produce motors exactly interchangeable with our competitors, always ready on stock which our customers particularly appreciate.



pag. 3



FLOW DIVIDERS TECHNICAL DATA

				Pressure			Flow per section		<u>_</u>			
MODEL		N° of pistons per section	Displacement per section	Max Cont.	Max Int.	Max	Max Continuous Max Int.		Max power per section		Dry Weight	
			cc/rev	bar	bar	bar	l/min	l/min	HP	kW	kg	
FD 20		9	20	250	280	320	25	35	12	9	55	
FD 27		9	27	250	280	320	35	50	16	12	55	
FD 34		9	34	250	280	320	45	60	20	15	55	
FD 50	FD	9	51	250	280	320	60	80	27	20	65	
FD 75		9	76	250	280	320	90	120	41	30	65	
FD 90		9	89	250	280	320	110	140	48	35	65	
FD 100		9	102	250	280	320	125	160	54	40	65	
FD 1 80		5	80	250	280	350	55	65	27	20	90	
FD 1 100	FD 1	5	100	250	280	350	70	85	34	25	90	
FD 1 150		5	157	250	280	350	110	130	52	38	90	
FD 2 250	ED 3	5	253	250	280	350	130	150	61	45	110	
FD 2 350	FD 2	5	362	250	280	350	165	190	79	58	110	
FD 3 400		5	397	250	280	350	170	190	82	60	180	
FD 3 450	ED 2	5	452	250	280	350	190	215	91	67	180	
FD 3 500	FD 3	5	491	250	280	350	210	230	101	74	180	
FD 3 600		5	594	250	280	350	230	260	109	80	180	
FD 4 800	ED 4	5	792	250	280	350	250	295	120	88	230	
FD 4 900	FD 4	5	904	250	280	350	285	335	139	100	230	
FD 5 1400		5	1376	250	280	300	290	340	119	87	460	
FD 5 1500	FD 5	5	1528	250	280	300	320	375	130	96	460	
FD 5 1600		5	1648	250	280	300	350	400	143	105	460	
FD 6 2500		5	2525	250	280	300	390	460	160	118	900	
FD 6 3000	ED 6	5	2983	250	280	300	440	525	179	132	900	
FD 6 3200	FD 6	5	3289	250	280	300	460	550	188	138	900	
FD 6 3500		5	3479	250	280	300	490	590	200	147	900	
FDM 34	FDM	9	34	250	280	320	45	60	20	15	35 per section	
FDM 50		9	51	250	280	320	60	80	27	20	45 per section	
FDM 75		9	76	250	280	320	90	120	48	35	45 per section	
FDM 100		9	102	250	280	320	125	160	54	40	45 per section	



HYDRAULIC FLUIDS RECOMMENDATIONS

HYDRAULIC FLUIDS

We recommend the use of hydraulic oils with anti-wear additives (ISO HM or HV) and minimum viscosity index of 95. Once normal working temperature is reached, oil viscosity must be at least 44 cSt, preferably in the range from 50 to 80 cSt

Hydraulic oils meeting Denison MF-O, Vickers M-2952-S I - 286-S performance requirements and DIN 51524 specifications, are preferred.

Pay particular attention if you use HE type oils (ecological fluid) because them can influence the motor seals compatibility, the motor performance and life. Please ask us for advice in case of HE type oils usage.

Mineral hydraulic oils are divided into four main types, designated by the International Standards Organisation (ISO) as HH, HL, HM and HV. We advise to use only products with HM or HV specifications.

HM type

These are the most widely employed hydraulic oils. They include small quantities of anti-wear additives to provide significant improvement in wear reduction. "Superior" quality HM type oils can be used for all equipment, with the added assurance that they will be suitable for the highest temperature.

HV type

HV hydraulic oils show minimal change in viscosity with temperature variations.

OIL VISCOSITY RECOMMENDATION

Room temperature HM type ISO-VG

- -20°C / 0°C BP ENERGOL HLP HM 22
- -15°C /+5°C
 BP ENERGOL HLP HM 32
 -8°C /+15°C
 BP BNERGOL HLP HM 46
- 0°C /+22°C BP ENERGOL HLP HM 68
- +8°C /+30°C BP ENERGOL HLP HM100
- -20°C /+5°C BP BARTRAN HV 32
 -15°C /+22°C BP BARTRAN HV 46
- 0°C /+30°C BP BARTRAN HV 68

Our motors have been designed to work also with:

- oils type ATF (Automatic Transmission Fluid)
- oils with viscosity SAE 10W 20 -30
- multigrade motor oils SAE 10 W/40 or 15 W/40
- universal oils

During cold start-up, avoid high-speed operation until the system is warmed up to provide adequate lubrication.

Continuous working temperature must not exceed 70°C. Every 5-8°C of increase from the optimum working temperature, the hydraulic fluid life decrease of about 40-50% (see OXIDATION). Therefore the motor life will be affected by consequence.

FIRE RESISTANT OIL LIMITATIONS

	Max cont.	Max int.	Max
	pressure	pressure	speed
HFA, 5-95% oil-water	103	138	50%
HFB, 60-40% oil-water	138	172	100%
HFC, water-glycol	103	138	50%
HFD ester phosphate	250	293	100%

FILTRATION

Hydraulic systems oil must always be filtered.

The choice of filtration grade derives from needs of service life and money spent. In order to obtain stated service life it is important to follow our recommendations concerning filtration grade.

When choosing the filter it is important to consider the amount of dirt particles that filter can absorb and still operate satisfactorily. For that reason we recommend filters showing when you need to substitute filtering cartridge.

- 25 µm filtration required in most applications
- 10 µm filtration in closed circuit applications

OXIDATION

Hydraulic oil oxidizes with time of use and temperature. Oxidation causes changes in colour and smell, acidity increase or sludge formation in the tank. Oxidation rate increases rapidly at surface temperatures above 60°C, in these situations oil should be checked more often. Every 5-8°C of increase from the optimum working temperature, the hydraulic fluid life decrease of about 40-50%.

The oxidation process increases the acidity of the fluid; the acidity is stated in terms of the "neutralization number". Oxidation is usually slow at the beginning and then it increases rapidly.

A sharp increase (by a factor of 2 to 3) in neutralization number between inspections shows that oil has oxidized too much and should be replaced immediately.

WATER CONTENT

Oil contamination by water can be detected by sampling from the bottom of the tank. Most hydraulic oils repel the water, which then collects at the bottom of the tank. This water must be drained off at regular intervals. Certain types of transmission oils and engine oils emulsify the water; this can be detected by coatings on filter cartridges or a change in the colour of the oil. In such cases, obtain your oil supplier advice.

DEGREE OF CONTAMINATION

Heavy contamination of the oil causes wear rising in hydraulic system components. Contamination causes must be immediately investigated and remedied.

ANALYSIS

In optimum operating conditions, we recommend to perfor an oil analysis 6 months. The analysis should cover viscosity, oxidation, water content, additives and contamination. Most oil suppliers are equipped to analyze oil state and to recommend appropriate action. Oil must be immediately replaced if the analysis shows that it is exhausted.



INSTRUCTIONS AND ADVICES

GENERAL **INFORMATION FLOW DIVIDERS**

If several motors or cylinders are operate from a single pump without any means of controlling their individual flows only the motor or cylinder with the lowest pressure requirement would start its work cycle. The motor or cylinder with the next lower pressure requirement would only start when the first unit has completed its work cycle. This mode of operation is normally undesiderable, the total pump flow must therefore be divided into a series of partial flows. This can be obtained in the following ways:

- Throttles or flow regulation valves in each motor or cylinder connection:
- Operating the units in a series circuit;
- By the flow dividers utilisation.

INSTALLATION

Hoses and piping must be clean and free from contamination. No other special requirements are necessary.

- Intermot always advices the application datasheet compilation to help the customer to perform the best Intermot flow divider type selection
- Flow divider can be mounted in any position
- In all the working conditions Intermot suggests the valve block version selection
- Consult factory for intermittent applications

START UP

The flow divider case and pistons must be completely filled with oil before starting. Infact, the drain line piping must be installed in a way that doesn't allow that the flow divider can be empty. Due to the low drain quantity it is absolutely necessary fill completely the flow divider before the start-up (this is fundamental for the first start-up especially). Do not load the flow divider immediately at the maximum pressure and speed but increase the load gradually at start-up. Especially when the flow divider has a small displacement, do not start the flow divider when the outlet pressure is already at the maximum value.

CASE DRAIN - CASE PRESSURE

The flow dividers have many drain ports that are internally connected together. At least one of these drain ports must be connected to the tank. For continuous use flushing is required to maintain the operating fluid viscosity between the recommended values (see Hydraulics fluid recommendations). The case pressure must not exceed 20 bar, in order to avoid the mechanical parts failure and flow divider leakage.

TEMPERATURE

Refer to hydraulic fluid recommendations.

VISCOSITY

Refer to hydraulic fluid recommendations.

HIGH PRESSURE APPLICATIONS

In case of high pressure applications, a Nitemper treatment on motor body or in cylinders it is suggested to increase wear and tear resistance.

MINIMUM FLOWS

The minimum recommended flow per section can be calculated as the 20% of the maximum continuous flow (refer to the flow dividers technical data).

SYNCHRONISATION

Intermot flow dividers are of the radial piston design. Thanks to this characteristic, they can reach a very high accuracy.

The synchronisation error depends by different factors:

- Oil viscosity and temperature:
- System pressure mean value and variability;
- Total flow rate that must be divided.

The approximate synchronisation error is about $\pm 0.5\%$. This synchronisation degree is maintained over a wide flow range. In addition, due to the low leakage rate located in every individual section, the high syncronisation degree is maintained with different load conditions.

The flow dividers are not able to compensate the errors caused by different oil compression deriving from different cylinder loads: therefore it is recommended minimize the oil volumes between the flow divider and the hydraulic actuators by the usage of short pipes (of the same diameter) of equal length foe each actuator. If this is not enough the synchronization error can be reduced or eliminated by an adequate use of relief pressure valves.

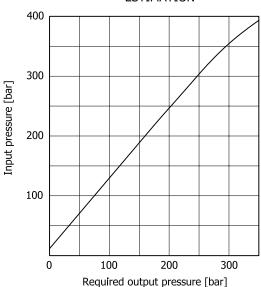
SYNCHRONIZATION RATIO

In general, the synchronization ratio is 1:1. Other synchronization ratios are possible, to obtain more details please contact Intermot technical departement.

INTERNAL PRESSURE DROP

The minimum recommended flow per section can be calculated as the 20% of the maximum continuous flow (refer to the flow dividers technical data). To minimize the internal pressure drop, that is higher when the flow divider is working at high working pressures, basing on the fact that often the flow divider works in only one direction, the flow divider can be realized, for example, in the return line. Another possibility to reduce the pressure drop, when the synchronization is required in the low pressures working phase only, the flow divider can be by-passed during the high pressure working phase by using check valves.

APPROXIMATED INTERNAL PRESSURE DROP **ESTIMATION**



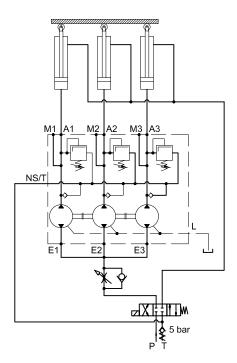


VALVES

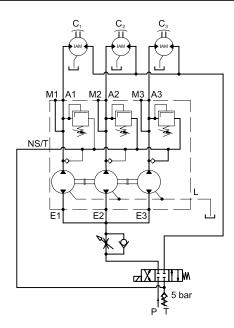
Intermot can provide balancing valves directly assembled on Intermot flow dividers. These valves are assembled together by the utilization of steel pipes supplied by Intermot to minimize the piping and fitting problems (see the next pages for more details).

INSTALLATION CIRCUIT

The use of flow dividers is not complicate if the hydraulic circuit has an appropriate design. Additional installation operations are not necessary. See even the internal pressure drop paragraph for more details.



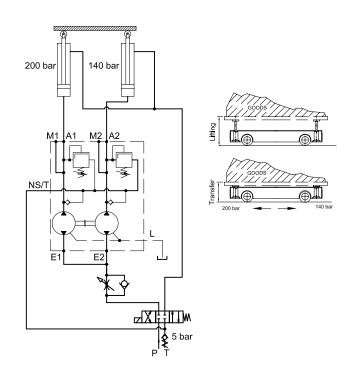
Flow divider application to control three cylinders



Flow divider application to control three motors

In some special conditions, for example when the load is not sufficiently high to create the minimum pressure difference that is needed to win the flow divider internal pressure, the user can add a further section (with equal or biggier displacement than the other sections) with the return line of this section directly connected to the tank.

With reference at the application example circuits, the check valves assure a minimum pressure in each flow divider section (in other words them act as anticavitation valves). The M1, M2 and M3 ports can be used as manometer ports. Refer to page 29 to obtain more details about the valve data and circuit.



Example: flow divider application circuit on a lifting truck



FORMULAS

TORQUE (1)

TORQUE (2)

POWER (1)

POWER (2)

SPEED

REQUIRED MOTOR DISPLACEMENT

REQUIRED PUMP FLOW RATE

Torque = (specific torque) · (pressure)

 $displacement \ [cc/rev] \cdot pressure \ [bar]$ Torque [Nm] =

Power [kW] = $\frac{\text{Torque}[Nm] \cdot \text{speed}[rpm]}{\text{Torque}[Nm] \cdot \text{speed}[rpm]}$

9549

Torque [Nm] · speed [rpm] Power [CV] =

 $\underline{\text{flow rate}} \, [\text{I/min}] \cdot 1000$ speed [rpm] =

displacement [cc/rev]

displacement [cc/rev] = $\frac{\text{max required torque [Nm]} \cdot 62.8}{\text{constant}}$

max pressure [bar]

flow [I/min] = $\frac{\text{displacement} [cc/rev] \cdot max speed}{\text{rpm}}$

1000

CONVERSIONS

LENGTH	1	m	=	39.3701	in		1	lbf	=	0.4536	kgf
			=	3.2808	ft	•			=	4.448	N
			=	1.0936	yd	•					
			=	1000	mm	PRESSURE	1	bar	=	14.223	psi
	1	in	=	0.0833	ft				=	0.99	atm
			=	25.4	mm	_			=	1.02	ata
	1	ft	=	0.3048	m				=	100000	Pa
			=	0.3333	yd					100	kPa
			=	12	in	_			=	0.1	MPa
	1	yd	=	0.9144	m	_	1	psi	=	0.0703	bar
			=	3	ft				_		
			=	36	in	FLOW	1	l/min	E	0.264	gpm
	1	km	=	1000	m	-			_=_		cc/min
			=	1093.6	yd	-	1	gpm	=	3.785	
			=	0.6214				_	_=_	3785	cc/min
	1	mile	=	1.609	km		1	m³/s	Ŀ	60000	l/min
			_ =	1760	yd				=	15852	gpm
								2			
MASS	_1		=	2.2046		VOLUME		m ³	=	1000	
	1	lb	=	0.4536	kg		1		=	61,023	
		,	1					. 2	_=_	0,264	
SPEED	1	m/s	=		km/h	-	1	in ³	=	0,01639	
				2.237		-			_=_	16,39	
				3.2808		-		11.10		0,004326	galUS
	1	km/h	=	0.2778		-	1	galUS	E	3,7879	<u> </u>
				0.6214		2011/22		1347	=	231,15	
	-1		=	0.9113	ft/s	POWER	1	kW	Ē	1.341	HP
	1	mph	=	1.609	km/h		1	LID	=	1.3596	CV
				0.447	m/s		1	HP	=	0.7457	Kw
	-1	£/a	=	1.467	ft/s	-			=	1.0139	CV
	1	ft/s	=	0.3048	m/s	TOPOUE	1	Nima		0.102	Lam
				1.0973 0.6818	km/h	TORQUE	1	Nm	=	0.102 0.7376	
FORCE	1	N	=_		mph	-	1	kam		9.806	
PURCE	1	IN	=	0.102	kgf	-	1	kgm	트	7.2325	Nm lbf ft
	1	kaf			lbf		1	lhf ft	=		
		kgf	<u> </u>	2.205 9.806	lbf N	-	1	lbf ft	Ē	0.1383 1.3558	kgm
			=	9.000	IN				=	1.3538	Nm